Towards Realization of

Computational Homogenization in Practice¹

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Abstract

We present a computational homogenization approach for linear and nonlinear solid mechanics problems, which is fully compatible with conventional finite element code architecture. A seamless implementation in ABAQUS is presented including Python script, validation problems and a web-link where script files, user-defined subroutines and input files can be accessed. For linear problems, we demonstrate how to utilize ABAQUS existing facilities to develop analysis attributes required for solving a unit cell problem. For nonlinear problems, a Python script invoked by a coarse scale stress update procedure is introduced to carry out the scale bridging. The purpose of this paper is twofold: (i) to motivate practitioners to adopt the computational homogenization as an integral part of their analysis and design process; and (ii) to encourage commercial code vendors to seamlessly integrate the architectures proposed in their legacy codes.

1. Introduction

Composite materials evolved from humble beginnings, such as ancient mud bricks reinforced with straw, 7000-year old bitumen covered reed boats, and American Indian wood and mud structures to premier man-made building blocks in modern-day society. Today, composite materials are used increasingly in high-performance applications that

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require high specific strength and/or stiffness, low electrical conductivity, transparency to radio emissions, resistance to corrosion, etc. According to the E-Composites research study [1], the aerospace industry alone is estimated to use \$4.6 billion worth of composite materials during 2005-2010. During this period, the global end product market for composites is projected to reach \$27 billion.

Numerous theories have been developed to predict the behavior of composite materials. Starting from the rule of mixtures dating back to the Renaissance era to various effective medium models of Eshelby [2], Hashin [3], Mori and Tanaka [4], self-consistent approaches of Hill [5] and Christensen [6] among many others to various mathematical homogenization methods pioneered by Bensoussan [7] and Sanchez-Palencia [8]. Computational aspects of homogenization have been an active area of research starting with a seminal contribution of Guedes and Kikuchi [9] for linear elasticity problems. Over the past decade major contributions have been made to extending the theory of computational homogenization to nonlinear regime [10, 11, 12, 13] and to improving fidelity and computational efficiency of numerical simulations [14,15,16,17,18,19, 20,21, 22, 23,24].

Today, computational homogenization technologies are rapidly maturing with computational efficiency remaining an outstanding issue. Yet, the adoption of these technologies by industry is at an embryonic stage. Historically, industry adopts a new (computational) technology only after it perceives solid evidence that it can shorten time-to-market cycle of a product or process. Over the past 15 years, computational homogenization technologies have been successfully verified and validated; and while industry have abandoned in-house finite element code development efforts and generally shied away from "academic" codes, commercial finite element software vendors have been slow to providing these capabilities. The main reason is not in lack of maturity as one may expect, but in perceived need for entirely new data structures that cannot be accommodated within conventional finite element code architectures.

There are two critical issues that have to be addressed in order to integrate computational homogenization technologies into conventional finite element code architectures:

1. <u>Analysis attributes</u>: the need to develop and to seamlessly integrate new analysis attributes, such as multiple overall strain loadings, periodic boundary conditions, etc.

2. <u>Scale bridging mechanism</u>: the need to control the fine scale problem from the coarse scale analysis, including proper data transfer and manipulation between the scales.

In this paper, we demonstrate how a two-scale analysis can be seamlessly carried out using ABAQUS for both linear and nonlinear solid mechanics problems. For linear problems, we demonstrate how to utilize ABAQUS existing facilities to develop analysis attributes required for solving a unit cell problem. For nonlinear problems, a Python script invoked by a coarse scale stress update procedure is introduced to carry out the scale bridging. A web-link is provided for user-defined subroutines, Python script and input files for use with ABAQUS code.

2. Linear computational homogenization

Using mathematical homogenization, a linear elastostatics problem with periodic coefficients can be decomposed (see Appendix for details and Remark 2 for nomenclature) into uncoupled fine and coarse scale problems:

a. Coarse scale problem

Find
$$u_i^c(\mathbf{x})$$
 on Ω such that :
 $\overline{L}_{ijmn} \varepsilon_{mn,x_j}^c + \overline{b}_i = 0$ on Ω
 $u_i^c = \overline{u}_i$ on Γ_u ; $\overline{\sigma}_{ij} n_j = \overline{t}_i$ on Γ_t
(1)

where \mathbf{x} and $\mathbf{y} = \mathbf{x} / \zeta$ are coarse and fine scale position vectors, respectively; $0 < \zeta \ll 1$; u_i^c is the coarse scale displacement; $\varepsilon_{mn}^c = u_{(m,x_n)}^c = \frac{1}{2} \left(\frac{\partial u_m^c}{\partial x_n} + \frac{\partial u_n^c}{\partial x_m} \right)$ the coarse scale strain; and \overline{b}_i the average unit cell body force. Summation convention is employed for repeated

indices.

b. Fine-scale (unit cell) problem

Find
$$\chi_{innn}(\mathbf{y})$$
 on Θ such that :

$$\begin{bmatrix} L_{ijkl} \left(\chi_{(k,y_l)mn} + I_{klmn} \right) \end{bmatrix}_{,y_j} = 0 \text{ on } \Theta$$

$$\chi_{imn}(\mathbf{y}) = \chi_{imn}(\mathbf{y} + \mathbf{Y}) \text{ on } \partial\Theta$$

$$\chi_{imn}(\mathbf{y}) = 0 \text{ on } \partial\Theta^{vert}$$
(2)

where $I_{klmn} = (\delta_{mk}\delta_{nl} + \delta_{nk}\delta_{ml})/2$; Θ the domain of the unit cell; $\partial \Theta^{vert}$ the vertices of the unit cell; and \overline{L}_{ijmn} the homogenized constitutive tensor components given as

$$\overline{L}_{ijmn} = \frac{1}{|\Theta|} \int_{\Theta} \sigma_{ij}^{mn} d\Theta$$
(3)

where $\sigma_{ij}^{mn}(\mathbf{y})$ are stress influence functions (i.e., stress induced by an overall unit strain ε_{mn}^{c}) defined as

$$\sigma_{ij}^{mn} = L_{ijkl} \left(\chi_{(k,y_l)mn} + I_{klmn} \right)$$
(4)

Finite element discretization of the coarse and fine scale fields, $u_i^c = N_{iA}^c(\mathbf{x})d_A^c$, $\chi_{mnk} = N_{kA}^f(\mathbf{y})d_{mnA}^f$, respectively, gives the two-scale matrix equations:

a. Discrete coarse-scale problem

$$\underbrace{\int_{\Omega} B_{ijA}^{c} \overline{L}_{ijkl} B_{klB}^{c} d\Omega}_{K_{AB}} \cdot d_{B}^{c} = \underbrace{\int_{\Omega} N_{iA}^{c} \overline{b}_{i} d\Omega}_{F_{A}} + \underbrace{\int_{\Gamma_{t}} N_{iA}^{c} \overline{t}_{i} d\Omega}_{F_{A}}$$

$$\underbrace{\int_{\Omega} B_{ijA}^{c} \overline{L}_{ijkl} B_{klB}^{c} d\Omega}_{K_{AB}} \cdot d_{B}^{c} = \underbrace{\int_{\Omega} N_{iA}^{c} \overline{b}_{i} d\Omega}_{F_{A}} + \underbrace{\int_{\Gamma_{t}} N_{iA}^{c} \overline{t}_{i} d\Omega}_{F_{A}}$$

$$\underbrace{\int_{\Omega} B_{ijA}^{c} \overline{L}_{ijkl} B_{klB}^{c} d\Omega}_{F_{A}} \cdot d_{B}^{c} = \underbrace{\int_{\Omega} N_{iA}^{c} \overline{b}_{i} d\Omega}_{F_{A}} + \underbrace{\int_{\Gamma_{t}} N_{iA}^{c} \overline{t}_{i} d\Omega}_{F_{A}}$$

$$\underbrace{\int_{\Omega} B_{ijA}^{c} \overline{L}_{ijkl} B_{klB}^{c} d\Omega}_{F_{A}} \cdot d_{B}^{c} = \underbrace{\int_{\Omega} N_{iA}^{c} \overline{b}_{i} d\Omega}_{F_{A}} + \underbrace{\int_{\Gamma_{t}} N_{iA}^{c} \overline{t}_{i} d\Omega}_{F_{A}} + \underbrace{\int_{\Gamma_{t}} N_{iA}^{c} \overline{t}_{i$$

b. Discrete fine-scale problem

$$\underbrace{\int_{\Theta} B_{ijA}^{f} L_{ijkl} B_{klB}^{f} d\Theta}_{K_{AB}} \cdot d_{mnB}^{f} = \underbrace{-\int_{\Theta} B_{ijA}^{f} L_{ijmn} d\Theta}_{F_{mnA}} \quad on \ \Theta$$

$$d_{mnB}^{f} \ periodic \quad on \ \partial\Theta; \ d_{mnB}^{f} = 0 \ on \ \partial\Theta^{vert}$$
(6)

where the subscript *A* denotes degrees-of-freedom; superscripts *c* and *f* denote coarse and fine scale fields; $B_{ijA}^f = N_{(i,x_j)A}^f$ and $B_{ijA}^c = N_{(i,x_j)A}^c$ are symmetric gradients of the corresponding shape functions.

In the following, we focus on implementation of the two-scale analysis in a commercial package of choice. Examples are given for implementation in ABAQUS. The two-scale linear elasticity analysis consists of the following four steps:

- 1. Solve a unit cell problem with multiple right hand side (RHS) vectors (Eq. (6)) and compute the stress influence functions;
- 2. Evaluate the overall constitutive tensor components \overline{L}_{ijmn} by Eq. (3);
- 3. Solve the coarse-scale problem; and
- 4. Postprocess stresses in critical (or all) unit cells

We start with Step 1, solution of a unit cell problem subjected to multiple RHS vectors (six in 3D due to symmetry of indices *mn*). In the matrix implementation, L_{ijmn} is a 6×6 matrix where *ij* represents six rows and *mn* six columns. Each column in L_{ijmn} can be extracted by multiplying L_{ijmn} with a unit overall strain, $\varepsilon_{mn}^{c} = 1$. For implementation in a commercial package, it is convenient to select ε_{mn}^{c} in the form of a unit thermal strain as

$$\varepsilon_{mn}^{c} = \kappa_{mn} \cdot \Delta T \tag{7}$$

where κ_{mn} and $\Delta T = 1$ are appropriately chosen thermal expansion coefficients and a unit temperature change. Multiple RHS vectors in Eq. (6) can be imposed by changing the thermal expansion coefficient for each loading case as shown in Table I. One way to accomplish it, is by submitting six separate jobs with different values of κ_{mn} .

Indices (mn)	11	22	33	23	13	12
thermal expansion coefficient vector $\mathbf{\kappa}$	$\left(1\right)$	$\begin{bmatrix} 0 \end{bmatrix}$	$\begin{bmatrix} 0 \end{bmatrix}$	$\begin{bmatrix} 0 \end{bmatrix}$	$\left[0 \right]$	$\left[0 \right]$
	0	1	0	0	0	0
	0	0	1	0	0	0
]0[
	0	0	0	0	1	0
	0	0	0	0	0	[1]

Table I: Thermal expansion coefficients (m,n) corresponding to six loading cases in 3D

Many commercial finite element packages, however, allow to defining temperaturedependent thermal conductivity that would permit a unit cell solution for multiple RHS vectors in a single job. In ABAQUS, this can be accomplished via *linear perturbation step*, which unlike the *general step* has no effect on subsequent steps. Due to temperature-dependence, for each step it is necessary to redefine the temperature change that would correspond to a different value of thermal conductivity. In ABAQUS, temperature-dependent thermal conductivity is defined in the user-defined subroutine UEXPAN().

We now turn to Step 2, an implementation of the overall constitutive tensor \overline{L}_{ijmn} through the integral in Eq. (3). Perhaps the most elegant implementation is by exploiting the relation between stress σ_{ij} , stress influence functions σ_{ij}^{mn} and overall constitutive tensor components \overline{L}_{ijmn} . Recall that the stress influence functions σ_{ij}^{mn} are stresses σ_{ij} obtained by subjecting a unit cell to a unit coarse-scale strain $\varepsilon_{mn}^{c} = 1$. In Step 1, we have already analyzed a unit cell subjected to six loading cases corresponding to precisely the same unit strains. Thus the stress influence functions σ_{ij}^{mn} are stress obtained for the aforementioned six loading cases. In ABAQUS, the model information and the output results for various loading cases are stored in ABAQUS output database (*job-name.odb*), which is a neutral binary file. Finally, we use a Python² script to access stress values at every integration point y_{I} in the unit cell stored in the database to numerically integrate Eq. (3)

$$\overline{L}_{ijmn} = \frac{1}{|\Theta|} \cdot \sum_{I=1}^{n_{int}} \sigma_{ij}^{mn}(y_I) J(y_I) W(y_I)$$
(8)

where $J(y_I)$ and $W(y_I)$ are Jacobian and weight, respectively, at an integration point positioned at y_I .

An important aspect of solving a unit cell problem is implementation of periodic boundary conditions. In ABAQUS, this can be accomplished via surface-to-surface constraints. In this type of a constraint, each node on a slave surface is constrained to

² Python is an object-oriented programming language that is widely used in the software industry, which is also the standard programming language for ABAQUS products

have the same motion as a closest point on the master surface. The following rules are used to form the master-slave relation. For an element-based master surface, a point on the master surface closest to a slave node is calculated and then used to determine the master node(s) that will be forming the constraint. For example, in Figure 1, nodes 1, 2 and 3 are used to constrain node *c*; nodes 1 and 2 constrain node *a*; and node 2 constrains node *b*. The element shape functions are used to set up the constraints. It is important to note that when master and slave surfaces have different mesh densities, the master surface should be chosen as the surface with a coarser mesh. Moreover, the POSITION TOLERANCE parameter should be set to be greater than the distance between two surfaces (see definition of ABAQUS keyword "*TIE" in online ABAQUS Keywords Reference Manual).



Figure 1: Master-slave relations for an element-based master surface

Finally, the coarse scale analysis is carried out using overall coefficients computed in Step 2. For the postprocessing (Step 4), the coarse scale strains obtained in Step 3 are used in combination with stress influence functions calculated in Step 1 to compute the fine scale stresses in critical (or all) unit cells as

$$\sigma_{ij} = \sigma_{ij}^{mn} \varepsilon_{mn}^{c} \tag{9}$$

For verification, we consider a three-dimensional fibrous unit cell. The phase properties of the microstructure are summarized in Table II. The unit cell is discretized with 351 tetrahedral elements as shown in Figure 2.

Materials	Young's Modulus	Poisson's ratio	Volume fraction
Titanium Matrix	68.9 GPa	0.33	0.733
SiC Fiber	379.2 GPa	0.21	0.267

Table II: Material properties for fibrous unit cell



Figure 2: Finite element mesh of a fibrous unit cell

Using Python script, the four steps are carried by submitting a single job. The visualization module of ABAQUS/CAE (ABAQUS/Viewer) can be used to output von Mises' stress σ^{mn} and displacement χ_{imn} influence functions as shown in Figure 3.



Figure 3: von Mises' Stress σ^{mn} and displacement χ_{imn} influence functions corresponding to mn components: (a) 11; (b) 22; (c) 33; (d) 23; (e) 13; (f) 12;

The overall properties of the composite obtained in Step 2 are depicted in Table III. For comparison, the overall properties obtained using Self-Consistent Method (SCM) and Mori-Tanaka Method (MTM), are also shown in the Table III.

Table III: The coefficients for homogenized stiffness matrix of the fibreous composite

Indices	11	22	33	23	13	12

	140.3 (136.6/134.2)	57.3 (61.8/61.4)	57.7 (57.8/57.3)	0.0 (0.0/0.0)	0.0 (0.0/0.0)	0.1 (0.0/0.0)
FEM Results (GPa) (SCM/MTM)		140.0 (136.6/134.2)	57.6 (57.8/57.3)	0.0 (0.0/0.0)	0.0 (0.0/0.0)	0.1 (0.0/0.0)
			185.6 (185.7/185.6)	0.0 (0.0/0.0)	0.0 (0.0/0.0)	0.0 (0.0/0.0)
				39.5 (40.1/38.2)	0.0 (0.0/0.0)	0.0 (0.0/0.0)
		SYM.			39.4 (40.1/38.2)	0.0 (0.0/0.0)
						36.5 (37.4/36.4)

For the coarse scale analysis in Step 3, we consider a cantilever beam subjected to a uniform distributed load along the top edge. For comparison, a reference solution is obtained using a single scale finite element analysis on a fine mesh. Deformed meshes and von Mises' stresses at a critical unit cell (left bottom corner) are shown in Figure 4. The stresses in a unit cell are obtained by postprocessing in Step 4.



For convenience, all input files, user-defined subroutines and Python script for the above example can be can be found in <u>http://www.rpi.edu/~fishj/***</u>.

3. Nonlinear computational homogenization

The two-scale algorithm presented in Section 2 can be generalized to account for material and geometric nonlinearities as long as the periodicity assumption remains intact. The governing equations directly follow from Eq. (10) and the integral of Eq. (10) in Appendix. The resulting two-scale problem summarized below is two-way coupled:

a. <u>Coarse scale problem</u>

Find
$$u_i^c(\mathbf{x})$$
 on Ω such that :
 $\overline{\sigma}_{ij,x_j} + \overline{b}_i = 0$ on Ω
 $u_i^c = \overline{u}_i$ on Γ_u ; $\overline{\sigma}_{ij}n_j = \overline{t}_i$ on Γ_t
(10)

b. Fine-scale (unit cell) problem

Find
$$u_i^f(\mathbf{y})$$
 on Θ such that :
 $\sigma_{ij,y_j} = 0$ on Θ
Kinematics + contitutive equations (11)
 $u_i^f(\mathbf{y}) = u_i^f(\mathbf{y} + \mathbf{Y})$ on $\partial \Theta$
 $u_i^f(\mathbf{y}) = 0$ on $\partial \Theta^{vert}$

where σ_{ij} is Cauchy stress. Note that in the fine scale problem in Eq. (11) we do not specify the form of kinematical and constitutive relations, i.e. any kinematical and constitutive equations available in a commercial code of choice or specified in a user-defined subroutine can be utilized for this purpose.

The link between the two scales is established by the overall Cauchy stress $ar{\sigma}_{ij}$ as

$$\bar{\sigma}_{ij} = \frac{1}{|\Theta|} \int_{\Theta} \sigma_{ij} d\Theta \tag{12}$$

Following the same discretization scheme as for linear problems yields an algebraic system of nonlinear equations:

a. Discrete coarse-scale problem

Given
$$_{n+1}\overline{b}$$
, $_{n+1}\overline{t}$, $_{n}\mathbf{x}^{c}$, find $_{n+1}\Delta\mathbf{d}^{c}$ such that:
 $_{n+1}r_{A}^{c}\left(_{n+1}\Delta\mathbf{d}^{c}\right) \equiv \underbrace{\int_{\Omega}{}_{n+1}B_{ijA}^{c}{}_{n+1}\overline{\sigma}_{ij}\left(_{n}\mathbf{x}^{c}{},_{n+1}\Delta\mathbf{\epsilon}^{c}\right)d\Omega}_{n+1f_{A}^{fint}} - \underbrace{\left(\int_{\Omega}{}_{\Omega}N_{iA}^{c}{}_{n-1}\overline{b}_{i}d\Omega + \int_{\Gamma_{t}}{}_{N}N_{iA}^{c}{}_{n+1}\overline{t}_{i}d\Omega\right)}_{n+1f_{A}^{fint}} = 0$

$$(13)$$
 $_{n+1}\mathbf{d}^{c} = {}_{n+1}\overline{\mathbf{d}} \quad on \quad \Gamma_{u}$
 $n \leftarrow n+1, \quad \text{Go to the next load increment}$

b. Discrete fine-scale problem

Given
$$_{n+1}^{i+1}\Delta \mathbf{\epsilon}^{c}$$
, $_{n+1}^{i+1}\Delta \mathbf{\omega}^{c}$, $_{n}\mathbf{x}^{f}$, $_{n}\sigma_{ij}$, find $_{n+1}^{i+1}\Delta \mathbf{d}^{f}$ such that:
 $_{n+1}^{i+1}r_{A}^{f}\left(_{n+1}^{i+1}\Delta \mathbf{d}^{f}\right) \equiv \int_{\Theta}_{n+1}^{i+1}B_{ijA}^{f} \cdot _{n+1}^{i+1}\sigma_{ij}\left(_{n}\mathbf{\sigma},_{n+1}^{i+1}\Delta \mathbf{\epsilon}^{c},_{n+1}^{i+1}\Delta \mathbf{\omega}^{c},_{n+1}^{i+1}\Delta \mathbf{\omega}^{f}\right) d\Theta = 0$ on Θ
 $_{n+1}^{i+1}\Delta \mathbf{d}^{f}$ periodic on $\partial\Theta$; $_{n+1}^{i+1}\Delta \mathbf{d}^{f} = 0$ on $\partial\Theta^{vert}$

$$(14)$$

where $_{n+1}r_A^c$ and $_{n+1}^{i+1}r_A^f$ are fine and coarse scale residuals, respectively;

 ${}^{i}_{n}\Delta\varepsilon_{kl}^{c} = \left(\frac{\partial_{n}^{i}\Delta u_{k}^{c}}{\partial_{n+1/2}x_{l}} + \frac{\partial_{n}^{i}\Delta u_{l}^{c}}{\partial_{n+1/2}x_{k}}\right)/2, \quad {}^{i}_{n}\Delta\varepsilon_{kl}^{f} = \left(\frac{\partial_{n}^{i}\Delta u_{k}^{f}}{\partial_{n+1/2}y_{l}} + \frac{\partial_{n}^{i}\Delta u_{l}^{f}}{\partial_{n+1/2}y_{k}}\right)/2 \text{ are the coarse and fine scale strain increments, respectively, typically computed with respect$

the coarse and fine scale strain increments, respectively, typically computed with respect to the derivative of the respective coordinates at the mid-step [25, 26];

$$\int_{n}^{i} \Delta \omega_{kl}^{c} = \left(\frac{\partial_{n}^{i} \Delta u_{k}^{c}}{\partial_{n+1/2} x_{l}} - \frac{\partial_{n}^{i} \Delta u_{l}^{c}}{\partial_{n+1/2} x_{k}} \right) / 2, \quad \int_{n}^{i} \Delta \omega_{kl}^{f} = \left(\frac{\partial_{n}^{i} \Delta u_{k}^{f}}{\partial_{n+1/2} y_{l}} - \frac{\partial_{n}^{i} \Delta u_{l}^{f}}{\partial_{n+1/2} y_{k}} \right) / 2$$
are corresponding vorticities;
$$\int_{n+1}^{i+1} \Delta u_{k}^{c} = N_{iA}^{c} \int_{n+1}^{i+1} \Delta u_{k}^{c} = N_{kA}^{c} \int_{n+1}^{i+1} \Delta u_{k}^{c} = N_{kA}^{f} \int_{n+1}^{i+1} \Delta u_{k}^{f} = N_{kA}^{f} \int_{n+1}^{i+1} \Delta u_{A}^{f} \int_{n+1}^{i+1} \Delta u_{A}^{f$$

 $_{n+1}\mathbf{x}^{*}$ ($_{n}\mathbf{x}^{*}$, $_{n+1}\Delta\mathbf{c}^{*}$, $_{n+1}\Delta\mathbf{c}^{*}$) (for large deformation analysis only) once the Newton process on the coarse scale converged. No summation convention is employed for left superscripts and subscripts.

In the following, we describe a two-scale nonlinear algorithm fully compatible with commercial code architecture. The discrete coarse scale problem (13) is solved incrementally using Newton's method. To solve for the unknown coarse scale displacement increment, ${}^{i+1}_{n+1}\Delta d^c$, the coarse scale problem is informed with the overall

Cauchy stress $_{n+1}^{i+1}\overline{\sigma}_{ij}$ and the overall instantaneous constitutive tensor $_{n+1}^{i+1}\overline{L}_{ijmn}$. These two quantities are extracted from the corresponding fine scale fields as

$${}^{i+1}_{n+1}\overline{\sigma}_{ij} = \frac{1}{|\Theta|} \int_{\Theta} {}^{i+1}_{n+1}\sigma_{ij} \left({}_{n}\boldsymbol{\sigma} , {}^{i+1}_{n+1}\Delta\boldsymbol{\varepsilon}^{c} , {}^{i+1}_{n+1}\Delta\boldsymbol{\omega}^{c} , {}^{i+1}_{n+1}\Delta\boldsymbol{\varepsilon}^{f} , {}^{i+1}_{n+1}\Delta\boldsymbol{\omega}^{f} \right) d\Theta$$
(15)

$$_{n+1}^{i+1}\overline{L}_{ijmn} = \frac{1}{|\Theta|} \int_{\Theta} \int_{\Theta} \int_{n+1}^{n+1} \sigma_{ij}^{mn} \left({}_{n} \boldsymbol{\sigma} , \int_{n+1}^{i+1} \Delta \boldsymbol{\epsilon}^{c}, \int_{n+1}^{i+1} \Delta \boldsymbol{\omega}^{c}, \int_{n+1}^{i+1} \Delta \boldsymbol{\epsilon}^{f}, \int_{n+1}^{i+1} \Delta \boldsymbol{\omega}^{f} \right) d\Theta$$
(16)

where $_{n+1}^{i+1}\sigma_{kl}\left(_{n}\sigma, _{n+1}^{i+1}\Delta\varepsilon^{c}, _{n+1}^{i+1}\Delta\omega^{c}\right)$ and $_{n+1}^{i+1}\sigma_{ij}^{mn}\left(_{n}\sigma, _{n+1}^{i+1}\Delta\varepsilon^{c}, _{n+1}^{i+1}\Delta\omega^{c}\right)$ denote Cauchy stress and the instantaneous Cauchy stress influence functions, respectively. It is important to note that these quantities correspond to converged (equilibrated) unit cell solution, which may or may not represent the converged coarse scale solution.

In ABAQUS, the coarse scale analysis is controlled by a user-defined subroutine UMAT(). At each integration point, increment n+1 and iteration i+1, ABAQUS calls UMAT() routine to carry out the following operations:

1. Write $\binom{i+1}{n+1}\Delta \varepsilon^{c}$, $\frac{i+1}{n+1}\Delta \omega^{c}$ to an external data file;

2. Write current coarse scale element number, integration point number, increment and iteration numbers into an external file;

3. Invoke Python script to prepare and submit the corresponding unit cell job and to calculate the overall quantities $_{n+1}^{i+1}\overline{\sigma}_{ij}$ and $_{n+1}^{i+1}\overline{L}_{ijmn}$; and

4. Read ${}^{i+1}_{n+1}\overline{\sigma}_{ij}$ and ${}^{i+1}_{n+1}\overline{L}_{ijmn}$ from an external data file.

We now turn to the fine scale problem. It is important to note that for every coarse scale solution $\binom{i+1}{n+1}\Delta \varepsilon^c$, $\binom{i+1}{n+1}\Delta \omega^c$, the discrete unit cell problems (14) are solved using Newton's method. Once (independent) Newton' processes for all unit cell problems (14) converged, the Cauchy stresses and the Cauchy stress influence functions are computed to evaluate the overall Cauchy stress (15) and the overall instantaneous constitutive tensor (16). From the implementation point of view, one needs to address the following two issues: (i) solution of the unit cell problems; and (ii) evaluation of the stress influence functions.

We start with the first issue. A unit cell geometry (or geometries), finite element mesh, boundary conditions and material models are defined in the input file. One may use commercial code library of materials or define a new material using user-defined facilities. For the unit cell analysis in ABAQUS, this step should be defined as a *general step*.

Unit cell analyzes are carried out incrementally from the converged unit cell solutions (displacements and stresses), which are stored in the restart file (unit-cell-name.stt). In ABAQUS, this is accomplished using ABAQUS keyword "*RESTART, WRITE". Two sets of restart files are prepared for each unit cell. One that stores the information from the previous converged stress ${}_{n}\sigma_{ij}$; the second contains the information from the last iteration, which may or may not represent the converged solution for the next increment. Changing the increment number serves as an indication for the first restart file to be

overwritten by the second one. For continuation of analyzes in ABAQUS, a keyword "*RESTART, READ" is used.

The coarse scale strain increment, ${}^{i+1}_{n+1}\Delta\varepsilon^c_{mn}$, is imposed in the form of thermal strains

$${}^{i+1}_{n+1}\Delta\varepsilon^c_{mn} = \kappa_{mn} \cdot \Delta T \tag{17}$$

where the thermal expansion coefficient and temperature change are chosen as $\kappa_{mn} = {}_{n+1}^{i+1} \Delta \varepsilon_{mn}^{c}$ and $\Delta T = 1$. In ABAQUS, UEXPAN() is used to define the thermal conductivity.

Depending on material model, stress updates can be carried out in two steps. In step one, stresses are updated by subjecting a unit cell to thermal strains, $\kappa_{mn} \cdot \Delta T$, where $\kappa_{mn} = {}^{i+1}_{n+1} \Delta \varepsilon^c_{mn}$. This step may include both material and rotational stress updates and it utilizes algorithms available in the commercial package of choice. In step two, a unit cell is subjected to a constant rotational increment ${}^{i+1}_{n+1} \Delta \mathbf{R}^c$ computed from the incremental vorticity ${}^{i+1}_{n+1} \Delta \boldsymbol{\omega}^c$ using well-established procedures [25, 26, 27].

We now focus on the second issue, an evaluation of the instantaneous Cauchy stress influence functions $_{n+1}^{i+1}\sigma_{ij}^{mn}$. The stress influence functions are evaluated only once the converged (on the fine scale) unit cell solution is obtained. Let $_{n+1}^{i+1}L_{ijmn}(\mathbf{y})$ and $_{n+1}^{i+1}B_{ijA}^{f}(\mathbf{y})$ be components of the instantaneous material properties and symmetric gradient of the shape functions, respectively, both computed for the converged unit cell solution. The influence functions are then obtained by solving a linearized unit cell problem

$$\int_{\Theta} {}^{i+1}B_{ijA} {}^{f}{}^{i+1}L_{ijst} {}^{i+1}B_{stB} {}^{f}d\Theta \cdot {}^{i+1}{}_{n+1}\Delta d_{mnB}^{f} = -\int_{\Theta} {}^{i+1}B_{ijA} {}^{f}{}^{i+1}L_{ijst} d\Theta \quad on \quad \Theta$$

$${}^{i+1}{}_{n+1}\Delta d_{mnB}^{f} \quad periodic \quad on \quad \partial\Theta; \quad {}^{i+1}{}_{n+1}\Delta d_{mnB}^{f} = 0 \quad on \quad \partial\Theta^{vert}$$

$$(18)$$

for six RHS vectors. This step is similar to Step 1 in linear homogenization discussed in Section 2. It is important to emphasize, however, that Eq. (18) is a linear perturbation step with fixed ${}_{n+1}^{i+1}L_{ijmn}$ (see Remark below). The unit cell displacements ${}_{n+1}^{i+1}\Delta d_{mnB}^{f}$ obtained and the influence functions

$${}^{i+1}_{n+1}\hat{\sigma}^{mn}_{ij} = {}^{i+1}_{n+1}L_{ijkl} \left({}^{i+1}_{n+1}B^{f}_{mnB} {}^{i+1}_{n+1}\Delta d^{f}_{klB} + I_{klmn} \right)$$
(19)

have no influence on the unit cell solutions in subsequent increments and/or iterations of the coarse scale problem.

The stress influence functions computed in Eq. (19) are free of coarse scale incremental rotation ${}^{i+1}_{n+1}\Delta \mathbf{R}^c$. One may proceed to defining the corotational overall constitutive tensor ${}^{i+1}_{n+1}\hat{L}_{ijmn}$, which is free of the coarse scale incremental rotation

$${}^{i+1}_{n+1}\widehat{\overline{L}}_{ijmn} = \frac{1}{|\Theta|} \int_{\Theta} {}^{i+1}_{n+1}\widehat{\sigma}_{ij}^{mn} \left({}_{n}\mathbf{\sigma} , {}^{i+1}_{n+1}\Delta\mathbf{\epsilon}^{c}, {}^{i+1}_{n+1}\Delta\mathbf{\omega}^{c} = 0, {}^{i+1}_{n+1}\Delta\mathbf{\epsilon}^{f}, {}^{i+1}_{n+1}\Delta\mathbf{\omega}^{f} \right) d\Theta$$

followed by an appropriate rotation of ${}^{i+1}_{n+1}\hat{L}_{ijmn}$ to the global coordinate system.

The flowchart illustrating the key implementation steps including the reference to the source files (available online at <u>http://www.rpi.edu/~fishj/***</u>) and ABAQUS commands at each step are depicted in Figure 5.

<u>Remark 1:</u> For user-defined inelastic material models, the instantaneous material properties $_{n+1}^{i+1}L_{ijmn}$ are automatically stored in ABAQUS database, and therfore Eq. (19) can be computed as a *linear perturbation step*. For inelastic material models within ABAQUS library the instantaneous material properties $_{n+1}^{i+1}L_{ijmn}$ are not accessible to the user, and therefore, a linear perturbation step cannot be performed unless the converged step corresponds to elastic process (see [26] for definition of elastic and inelastic processes). In this case one has to carry out six *general perturbation steps* instead. A general perturbation step is defined with an infinitesimal increment of coarse scale strain and large convergence tolerance so that only a single iteration on the fine scale is carried out.



For verification, we consider several examples. The first two consider a coarse scale domain in the shape of a block discretized with a single brick element subjected to transverse tension. On the fine scale, we consider a fibrous unit cell introduced in Section 2 obeying perfect plasticity for the matrix phase ($\sigma^{Y} = 24 MPa$) and linear elasticity for the fiber phase.

The overall response of the composite is obtained using computational framework presented in this section. The overall stress-strain curve along the loading directions is shown in Figure 6 (the reference solution is obtained using a single scale analysis with a fine mesh). The relevant files for this example can be downloaded from http://www.rpi.edu/~fishj/***.



Figure 6: $\sigma_{xx} - \mathcal{E}_{xx}$ relation for the transverse tension problem with matrix obeying plasticity model

It is important to note that the algorithms presented here are independent of material model considered. For the second example, we keep the same geometry, mesh and boundary conditions, but model the matrix phase using continuum damage mechanics model with arctangent form of damage evolution law [27]. The resulting overall stress-strain curve is shown in Figure 7.



Figure 7: $\sigma_{xx} - \mathcal{E}_{xx}$ relation for the transverse tension problem with matrix obeying damage model

For the final example, we simulate crack propagation in a rectangular plate made of a woven composite microstructure [28]. A quarter of the plate is considered due to symmetry. A uniformly distributed tensile load is applied along the top edge. Continuum damage model considered in the previous example is employed. The value of damage parameter ($\omega = 1$ fully cracked) governs crack formation. Crack propagation obtained with the two-scale homogenization and the reference solution obtained with a single finite element mesh containing over one half million of elements are depicted in Figures 9 and 8, respectively. Figure 10 compares the crack length obtained by the two methods. The two-scale homogenization performs fairly well considering the fact that solution periodicity in the vicinity of the crack does not exist.



Figure 8: Crack propagation using single scale finite element analysis (reference solution)



Figure 9: Crack propagation in the coarse scale model



Figure 10: Crack length versus time

4. Summary and future research direction

We have demonstrated that computational homogenization is fully compatible with conventional finite element code architecture. Once all the input files, user-defined subroutines and Python script are in place, the two-scale analysis can be executed with a single "push button." We hope that the manuscript will motivate practitioners to adopt the computational homogenization as an integral part of the analysis and design process. We also hope that commercial code vendors will seamlessly integrate the architectures proposed in their legacy codes.

The issue of computational cost of the two-scale nonlinear analysis has not been addressed in the present manuscript. As one may expect, the cost is very high indeed. Let N_{cells} be the number of Gauss points in the coarse scale, n be the number of load increments in the coarse scale, I_{coarse} and I_{fine} be the average number of iterations in the coarse and fine scales respectively. Then the total number of linear solves of the fine scale problem is $N_{cells} \cdot n \cdot I_{coarse} \cdot I_{fine}$, certainly a formidable computational cost if the number of unit cells and degrees-of-freedom in a unit cell is substantial.

We believe that it is a combination of two remedies that will eventually reduce the computational cost to a manageable size. The first is utilization of parallel computing since unit cell computations are fully parallelizable. The second is coarse-graining or model reduction. Some (but not all) noteworthy efforts in this direction have been mentioned in the introduction of this paper [11-22]. In our upcoming manuscript [29], we will demonstrate how one specific model reduction approach can be seamlessly integrated into commercial finite element code architecture.

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Appendix: Mathematical homogenization for linear elasticity

The strong form of the boundary value problem for linear elastostatics is given by

$$\sigma_{ij,x_i}^{\zeta} + b_i = 0 \quad on \ \Omega^{\zeta} \tag{1}$$

$$\sigma_{ij}^{\zeta} = L_{ijkl}^{\zeta} \varepsilon_{kl}^{\zeta} \quad on \ \Omega^{\zeta}$$
⁽²⁾

$$\varepsilon_{kl}^{\zeta} \equiv u_{(k,x_l)}^{\zeta} = (u_{k,x_l}^{\zeta} + u_{l,x_k}^{\zeta})/2 \quad on \ \Omega^{\zeta}$$
(3)

$$\mu_i^{\zeta} = \overline{\mu}_i \quad on \ \Gamma_u \tag{4}$$

$$\sigma_{ij}^{\zeta} n_j = t_i \quad on \ \Gamma_t \tag{5}$$

Eqs. (1)-(5) are equilibrium equations, constitutive relations, kinematical relations, displacement boundary conditions and traction boundary conditions. The superscript ζ denotes Y-periodicity of the corresponding function f, i.e. $f(\mathbf{x}, \mathbf{y} + k\mathbf{Y}) = f(\mathbf{x}, \mathbf{y})$, where Y is the characteristic size of the fine scale (unit cell), and k is a non-zero integer.

A two-scale asymptotic expansion is employed to approximate the displacement field

$$u_i^{\zeta} = u_i^0(\boldsymbol{x}, \boldsymbol{y}) + \zeta u_i^1(\boldsymbol{x}, \boldsymbol{y}) + \zeta^2 u_i^2(\boldsymbol{x}, \boldsymbol{y}) + \cdots$$
(6)

where $u_i^{\Box}(\mathbf{x}, \mathbf{y})$ are Y-periodic functions, for $\Box = 0, 1, 2, ...$

Using the chain rule of the differentiation $f_{,x_i}(\mathbf{x}, \mathbf{y}) = f_{,x_i} + \zeta^{-1} f_{,y_i}$, and kinematical relations Eq. (3), the asymptotic expansion of the strain field is given as

$$\varepsilon_{kl}^{\zeta} = \zeta^{-1} \varepsilon_{kl}^{-1}(\boldsymbol{x}, \boldsymbol{y}) + \zeta^{0} \varepsilon_{kl}^{0}(\boldsymbol{x}, \boldsymbol{y}) + \zeta^{1} \varepsilon_{kl}^{1}(\boldsymbol{x}, \boldsymbol{y}) + \cdots$$
(7)

where,

$$\begin{cases} \varepsilon_{kl}^{-1} = \varepsilon_{kly}^{0} \\ \varepsilon_{kl}^{0} = \varepsilon_{klx}^{0} + \varepsilon_{kly}^{1} \quad and \quad \varepsilon_{klx}^{\Box} = u_{(k,x_{l})}^{\Box}; \varepsilon_{kly}^{\Box} = u_{(k,y_{l})}^{\Box} \quad for \Box = 0, 1, 2, \dots \\ \varepsilon_{kl}^{1} = \varepsilon_{klx}^{1} + \varepsilon_{kly}^{2} \end{cases}$$

The asymptotic expansion of the stress field is

$$\sigma_{ij}^{\zeta} = \zeta^{-1} \sigma_{ij}^{-1}(\boldsymbol{x}, \boldsymbol{y}) + \zeta^{0} \sigma_{ij}^{0}(\boldsymbol{x}, \boldsymbol{y}) + \zeta^{1} \sigma_{ij}^{1}(\boldsymbol{x}, \boldsymbol{y}) + \cdots$$
(8)

where for linear elastic problems $\sigma_{ij}^{\Box} = L_{ijkl} \varepsilon_{kl}^{\Box}$, for $\Box = -1, 0, 1, ...$

Inserting the asymptotic expansion of the stress field Eq.(8) into the equilibrium Eq.(1) and identifying terms with equal powers of ζ gives various order equilibrium equations

$$O(\zeta^{-2}) \to \sigma_{ij,y_j}^{-1} = 0$$
(9)

$$O(\zeta^{-1}) \to \sigma^0_{ij,y_j} + \sigma^{-1}_{ij,x_j} = 0$$
(10)

$$O(\zeta^0) \to \sigma^1_{ij, y_j} + \sigma^0_{ij, x_j} + b_i = 0$$
(11)

 $O(\zeta^{-2})$ equilibrium (Eq. (9)) is considered first. Premultiplying it by u_i^0 , integrating over the unit cell domain Θ and subsequently integrating by parts gives

$$\underbrace{\int_{\partial\Theta} u_i^0 \sigma_{ij}^{-1} n_j d\Gamma}_{=0, \ periodicity} - \int_{\Theta} u_{(i, y_j)}^0 L_{ijkl} u_{(k, y_l)}^0 d\Theta = 0$$
(12)

The first term in Eq. (12) vanishes due to periodicity. Moreover, assuming that L_{ijkl} is a positive definite yields

$$u_{(i,y_j)}^0 = 0 \Longrightarrow u_i^0 = u_i^0(\boldsymbol{x})$$
(13)

The $O(\zeta^{-1})$ equilibrium, Eq.(10), can be written as

$$\left[L_{ijkl}\left(\varepsilon_{klx}^{0}+\varepsilon_{kly}^{1}\right)\right]_{,y_{j}}=0$$
(14)

where we exploited $u_i^0 = u_i^0(\mathbf{x}) \Longrightarrow \sigma_{ij}^{-1} = 0$.

Introducing the decomposition of the second term in the asymptotic expansion of displacement field

$$u_i^1(\boldsymbol{x}, \boldsymbol{y}) = \chi_{imn}(\boldsymbol{y}) \varepsilon_{mnx}^0(\boldsymbol{x})$$
(15)

and taking into account arbitrariness of the coarse scale strain field $\varepsilon_{mnx}^0(\mathbf{x})$, the strong form of the unit cell problem for the influence function $\chi_{imn}(\mathbf{y})$ on Θ is given by

$$\left[L_{ijkl}\left(\chi_{(k,y_l)mn}+I_{klmn}\right)\right]_{,y_j}=0 \quad on \quad \Theta$$
(16)

$$\chi_{imn}(\mathbf{y}) = \chi_{imn}(\mathbf{y} + \mathbf{Y}) \quad on \quad \partial\Theta \tag{17}$$

$$\chi_{imn}(\mathbf{y}) = 0 \quad on \quad \partial \Theta^{vert} \tag{18}$$

where $I_{klmn} = (\delta_{mk}\delta_{nl} + \delta_{nk}\delta_{ml})/2$; $\partial \Theta^{vert}$ are the vertices of the unit cell. Eq.(18) is often replaced by the normalization condition $\int_{\Theta} \chi_{imn} d\Theta = 0$.

Finally, substituting constitutive relations for σ_{ij}^0 into equilibrium Eq.(11) yields

$$\sigma_{ij,y_j}^1 + L_{ijkl} \left(\chi_{(k,y_l)mn} + I_{klmn} \right) \varepsilon_{mnx,x_j}^0 + b_i = 0$$
⁽¹⁹⁾

Integrating Eq.(19) over the unit cell domain and accounting for the periodicity of σ_{ij}^1 yields

$$\overline{L}_{ijmn}\varepsilon^0_{mnx,x_j} + \overline{b}_i = 0 \tag{20}$$

where

$$\overline{L}_{ijmn} = \frac{1}{|\Theta|} \int_{\Theta} L_{ijkl} \left(\chi_{(k,y_l)mn} + I_{klmn} \right) d\Theta$$
$$\overline{b}_i = \frac{1}{|\Theta|} \int_{\Theta} b_i d\Theta$$

Remark 2: In Sections 2 and 3 we adopted the following nomenclature

$$\boldsymbol{\sigma}_{ij} \equiv \boldsymbol{\sigma}_{ij}^{0}, \quad \boldsymbol{\varepsilon}_{mn}^{c} \equiv \boldsymbol{\varepsilon}_{mnx}^{0}, \quad \boldsymbol{u}_{i}^{c} = \boldsymbol{u}_{i}^{0}, \quad \boldsymbol{u}_{i}^{f} = \boldsymbol{u}_{i}^{1}$$

which is more transparent to readers that are not familiar with nomenclature of the mathematical homogenization.

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