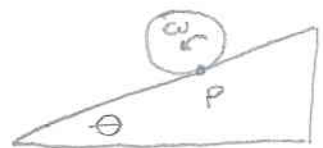


Important example

(156)



Find acceleration of cylinder of radius R and mass M rolling without slipping down the wedge.

Method 1 Point P has zero velocity

(no slipping) compute τ & L about

P : $L = I_P \omega$ $I_P = \frac{3}{2} MR^2$

note we are using a fixed axis thru P \perp to the page

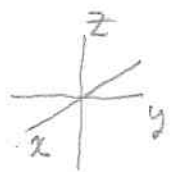
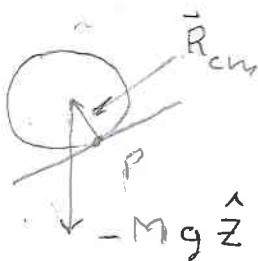
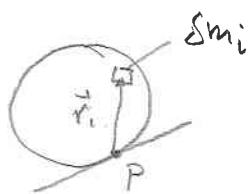
$$\vec{\tau} = \sum_i \vec{r}_i \times (-\delta m_i g \hat{z})$$

$$= - \sum_i \delta m_i \vec{r}_i \times \hat{z} g$$

$\vec{R}_{cm} M$

$$= - Mg \vec{R}_{cm} \times \hat{z}$$

$$= MgR \hat{x} \sin \theta$$



November 24, 2020

(157)

$$\frac{dL}{dt} = \tau \quad (\text{equate } \hat{x} \text{ components})$$

$$\frac{d(I\omega)}{dt} = MgR \sin \theta$$

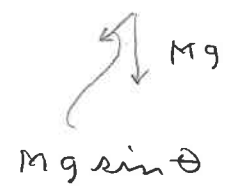
Conservation of L

$$\frac{3}{2} MR^2 \dot{\omega} = MgR \sin \theta$$

$$a_{cm} = R\dot{\omega} = \frac{2}{3} g \sin \theta$$

$< g \sin \theta$ result for block sliding on wedge

Method 2



Work with torques about cm :

$$I \dot{\omega} = FR$$

$$\frac{1}{2} MR^2 \frac{a}{R} = FR$$

$$Ma = Mg \sin \theta - F$$

$$= Mg \sin \theta - \frac{1}{2} Ma$$

$$\frac{3}{2} Ma = Mg \sin \theta$$

$$a = \frac{2}{3} g \sin \theta \checkmark$$

$$\frac{dL}{dt} = \tau \quad (\text{equate } \hat{x} \text{ components})$$

$$\frac{d(I\dot{\omega})}{dt} = MgR \sin \theta$$

$$\frac{3}{2} MR^2 \dot{\omega} = MgR \sin \theta$$

$$a_{cm} = R\dot{\omega} = \frac{2}{3} g \sin \theta$$

$< g \sin \theta$ result for
block sliding on wedge

Method 2



$$Mg \sin \theta$$

Work with torques about
cm:

$$I\dot{\omega} = FR$$

$$\frac{1}{2} MR^2 \frac{a}{R} = FR$$

$$Ma = Mg \sin \theta - F$$

$$= Mg \sin \theta - \frac{1}{2} Ma$$

$$\frac{3}{2} Ma = Mg \sin \theta$$

$$a = \frac{2}{3} g \sin \theta \quad \checkmark$$

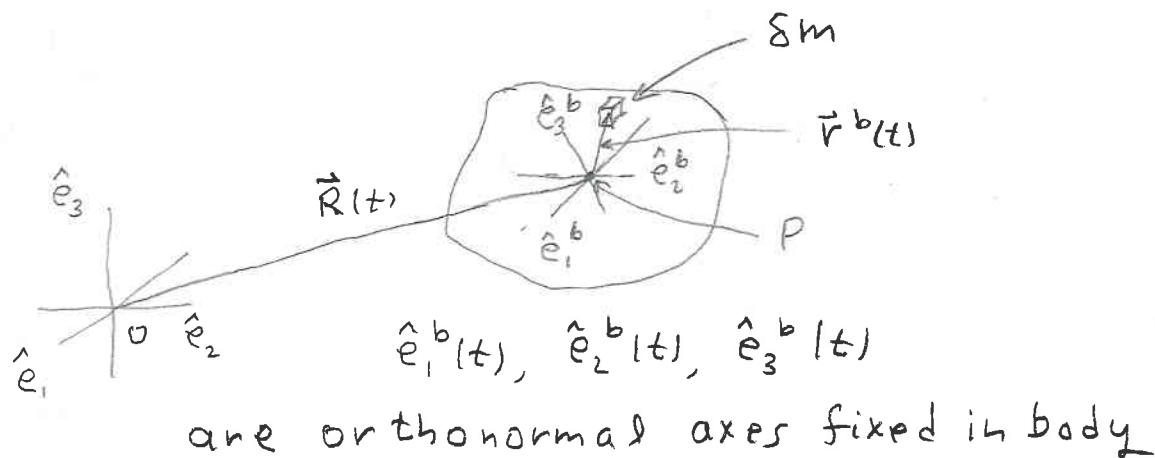
This method is also subtle

$$I\dot{\omega} = FR \quad \text{equate } \frac{dL}{dt} \text{ \& } \tau$$

computed about cm which is
an accelerating, non-inertial
system !!

This is OK since fictitious
force due to acceleration
exerts no torque about cm.

General motion of a rigid body
1. Angular velocity done better



This method is also subtle

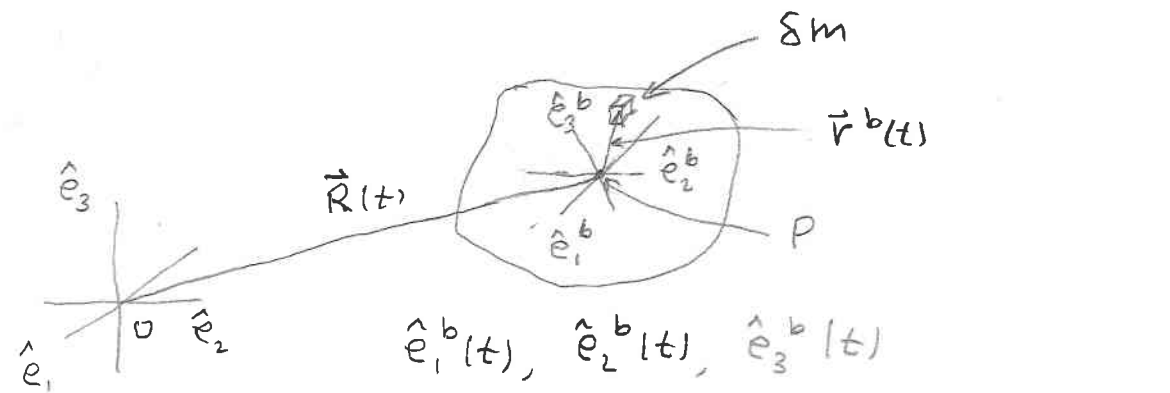
$$I \dot{\omega} = FR \text{ equate } \frac{dL}{dt} \neq \tau$$

computed about cm which is an accelerating, non-inertial system !!

This is OK since fictitious force due to acceleration exerts no torque about cm.

General motion of a rigid body

1. Angular velocity done better



are orthonormal axes fixed in body

The rotating axes fixed in the body are related to the fixed axes by a time dependent rotation matrix

$$\text{Recall } \hat{e}_i^b(t) = \sum_{j=1}^3 M_{ij}(t) \hat{e}_j$$

$$\hat{e}_j(t) = \sum_{i=1}^3 [M(t)]_{ji}^t \hat{e}_i^b(t)$$

$$\vec{v}_i(t) = \vec{R}(t) + \vec{v}_i^b(t)$$

$$\vec{v}_i(t) = \frac{d\vec{R}(t)}{dt} + \frac{d}{dt} \vec{r}_i^b(t)$$

$$\vec{r}_i^b(t) = \sum_{j=1}^3 (r_i^b)_j \hat{e}_j^b(t)$$

← rotating

↑ constant

$$\frac{d\vec{r}_i^b(t)}{dt} = \sum_{j=1}^3 (r_i^b)_j \frac{d\hat{e}_j^b(t)}{dt}$$

$$= \sum_{j=1}^3 (r_i^b)_j \left[\sum_k \frac{dM_{jk}}{dt} \hat{e}_k^b \right]$$

← $\sum_k (M^t)_{jk} \hat{e}_k^b$

The rotating axes fixed in the body are related to the fixed axes by a time dependent rotation matrix

Recall $\hat{e}_i^b(t) = \sum_{j=1}^3 M_{ij}(t) \hat{e}_j$
 $\hat{e}_j(t) = \sum_{i=1}^3 [M(t)]_{ji}^t \hat{e}_i^b(t)$

$\vec{r}_i(t) = \vec{R}(t) + \vec{r}_i^b(t)$

$\vec{v}_i(t) = \frac{d\vec{R}(t)}{dt} + \frac{d}{dt} \vec{r}_i^b(t)$

$\vec{r}_i^b(t) = \sum_{j=1}^3 (r_i^b)_j \hat{e}_j^b(t)$ ← rotating

↑ constant

$\frac{d\vec{r}_i^b(t)}{dt} = \sum_{j=1}^3 (r_i^b)_j \frac{d\hat{e}_j^b(t)}{dt}$
 $= \sum_{j=1}^3 (r_i^b)_j \left[\sum_k \frac{dM_{jk}}{dt} \hat{e}_k \right]$ ← $\sum_k (M^t)_{jk} \hat{e}_k$

$= \sum_{i=1}^3 (r_i^b)_i \left[\sum_{j=1}^3 \frac{dM_{ij}}{dt} \left(\sum_{k=1}^3 M_{jk}^t \hat{e}_k^b \right) \right]$
 $= \sum_{i=1}^3 \sum_{k=1}^3 (r_i^b)_i \underbrace{\left[\sum_{j=1}^3 \frac{dM_{ij}}{dt} M_{jk}^t \right]}_{\left(\frac{dM}{dt} M^t \right)_{ik}} \hat{e}_k^b$

But $M(t) M(t)^t = I$

$\therefore \frac{dM}{dt} M^t + M \frac{dM^t}{dt} = 0$

Or $\frac{dM}{dt} M^t = -M \frac{dM^t}{dt} = -\left(\frac{dM}{dt} M^t \right)^t$

$\Rightarrow \frac{dM}{dt} M^t = \begin{pmatrix} 0 & +\omega_3 & -\omega_2 \\ -\omega_3 & 0 & +\omega_1 \\ +\omega_2 & -\omega_1 & 0 \end{pmatrix}$

$\& \frac{d\vec{r}_i^b}{dt} = \begin{pmatrix} r_1^b & r_2^b & r_3^b \end{pmatrix} \begin{pmatrix} 0 & +\omega_3 & -\omega_2 \\ -\omega_3 & 0 & +\omega_1 \\ +\omega_2 & -\omega_1 & 0 \end{pmatrix} \begin{pmatrix} \hat{e}_1^b \\ \hat{e}_2^b \\ \hat{e}_3^b \end{pmatrix}$

$= (-\omega_3 r_2^b + \omega_2 r_3^b) \hat{e}_1^b + (\omega_3 r_1^b - \omega_1 r_3^b) \hat{e}_2^b$
 $+ (-\omega_2 r_1^b + \omega_1 r_2^b) \hat{e}_3^b$
 $= \vec{\omega} \times \vec{r}_i^b$

$$= \sum_{i=1}^3 (r^b)_i \left[\sum_{j=1}^3 \frac{dM_{ij}}{dt} \left(\sum_{k=1}^3 M_{jk}^t \hat{e}_k^b \right) \right]$$

$$= \sum_{i=1}^3 \sum_{k=1}^3 (r^b)_i \left[\sum_{j=1}^3 \frac{dM_{ij}}{dt} M_{jk}^t \right] \hat{e}_k^b$$

$$\underbrace{\left(\frac{dM}{dt} M^t \right)_{ik}}$$

But $M(t) M(t)^t = I$

$\therefore \frac{dM}{dt} M^t + M \frac{dM^t}{dt} = 0$

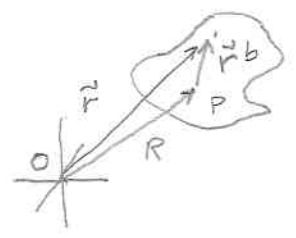
Or $\frac{dM}{dt} M^t = - M \frac{dM^t}{dt} = - \left(\frac{dM}{dt} M^t \right)^t$

$\Rightarrow \frac{dM}{dt} M^t = \begin{pmatrix} 0 & +\omega_3 & -\omega_2 \\ -\omega_3 & 0 & +\omega_1 \\ +\omega_2 & -\omega_1 & 0 \end{pmatrix}$

$\& \frac{d\vec{r}^b}{dt} = \begin{pmatrix} r_1^b & r_2^b & r_3^b \end{pmatrix} \begin{pmatrix} 0 & +\omega_3 & -\omega_2 \\ -\omega_3 & 0 & +\omega_1 \\ +\omega_2 & -\omega_1 & 0 \end{pmatrix} \begin{pmatrix} \hat{e}_1^b \\ \hat{e}_2^b \\ \hat{e}_3^b \end{pmatrix}$

$= (-\omega_3 r_2^b + \omega_2 r_3^b) \hat{e}_1^b + (\omega_3 r_1^b - \omega_1 r_3^b) \hat{e}_2^b$
 $+ (-\omega_2 r_1^b + \omega_1 r_2^b) \hat{e}_3^b$
 $= \vec{\omega} \times \vec{r}^b$

Thus motion of a rigid body can be described as translation of a point P fixed in the body and rotation about P



$\vec{r}(t) = \vec{R}(t) + \vec{r}^b(t), \quad \frac{d\vec{r}(t)}{dt} = \frac{d\vec{R}(t)}{dt} + \vec{\omega}(t) \times \vec{r}^b(t)$

Use this result to determine angular momentum about O:

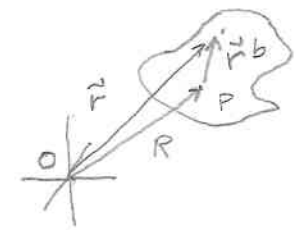
$\vec{L} = \sum_n [\vec{R}(t) + \vec{r}_n^b(t)] \times \left[\frac{d\vec{R}(t)}{dt} + \vec{\omega}(t) \times \vec{r}_n^b(t) \right] \delta m_n$

This is complicated unless we choose P = center-of-mass so $\sum_n \vec{r}_n^b(t) \delta m_n = 0$

Then $\vec{L} = \underbrace{\vec{R}(t) \times \frac{d\vec{R}(t)}{dt} M}_{\vec{L} \text{ of com about O}} + \underbrace{\sum_n \vec{r}_n^b \times (\vec{\omega} \times \vec{r}_n^b) \delta m_n}_{\vec{L} \text{ of body about the center of mass}}$

$L = \vec{R}(t) \times \left(M \frac{d\vec{R}(t)}{dt} \right) + \sum_{a,a'} \hat{e}_a^b(t) I_{aa'} \omega_{a'}^b(t)$

Thus motion of a rigid body can be described as translation of a point P fixed in the body and rotation about P



$$\vec{r}(t) = \vec{R}(t) + \vec{r}^b(t), \quad \frac{d\vec{r}(t)}{dt} = \frac{d\vec{R}(t)}{dt} + \vec{\omega}(t) \times \vec{r}^b(t)$$

Use this result to determine angular momentum about O:

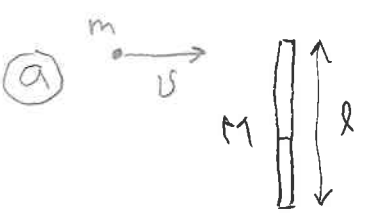
$$\vec{L} = \sum_n [\vec{R}(t) + \vec{r}_n^b(t)] \times \left[\frac{d\vec{R}(t)}{dt} + \vec{\omega}(t) \times \vec{r}_n^b(t) \right] \delta m_n$$

This is complicated unless we choose P = center-of-mass so $\sum_n \vec{r}_n^b(t) \delta m_n = 0$

Then
$$\vec{L} = \underbrace{\vec{R}(t) \times \frac{d\vec{R}}{dt} M}_{\vec{L} \text{ of com about } O} + \underbrace{\sum_n \vec{r}_n^b \times (\vec{\omega} \times \vec{r}_n^b) \delta m_n}_{\vec{L} \text{ of body about the center of mass}}$$

$$L = \vec{R}(t) \times \left(M \frac{d\vec{R}}{dt} \right) + \sum_{a,a'} \hat{e}_a^b(t) I_{aa'} \omega_{a'}^b(t)$$

2. Work out two examples

a)  mass m moving with velocity $v \perp$ to rod of length l + mass M (at rest) strikes and sticks to the end of the rod. Describe the resulting motion.

* CM motion moves with $(m+M)\vec{V}_{cm} = m\vec{v}$
 or $\vec{V}_{cm} = \frac{m}{m+M} \vec{v}$

Note location of cm must be d above center where $(m+M)d = m\frac{l}{2} + M \cdot 0 \Rightarrow d = \frac{m}{m+M} \frac{l}{2}$

* Rotation about CM

$$L_{int} = m v \left(\frac{l}{2} - d \right) = \frac{m M v}{m+M} \frac{l}{2}$$

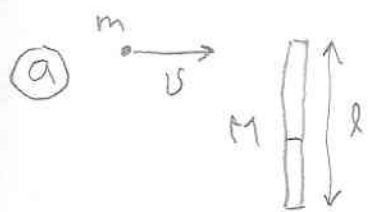
$$L_{final} = I_{cm} \omega$$

$$I_{cm} = \underbrace{m \left(\frac{l}{2} - d \right)^2}_{I \text{ for } m} + \underbrace{\frac{1}{12} M l^2 + M d^2}_{\text{parallel axis theorem for rod about new CM}}$$

parallel axis theorem for rod about new CM

2. Work out two examples

162



mass m moving with velocity $v \perp$ to rod of length l + mass M (at rest)

strikes and sticks to the end of the rod. Describe the resulting motion.

* CM motion moves with $(m+M)\vec{V}_{cm} = m\vec{v}$
or $\vec{V}_{cm} = \frac{m}{m+M} \vec{v}$

Note location of cm must be d above center where $(m+M)d = m\frac{l}{2} + M \cdot 0 \Rightarrow d = \frac{m}{m+M} \frac{l}{2}$

* Rotation about CM

$$L_{int} = m v \left(\frac{l}{2} - d \right) = \frac{m M v}{m+M} \frac{l}{2}$$

$$L_{final} = I_{cm} \omega$$

$$I_{cm} = \underbrace{m \left(\frac{l}{2} - d \right)^2}_{I \text{ for } m} + \underbrace{\frac{1}{12} M l^2 + M d^2}_{\text{parallel axis theorem for rod about new CM}}$$

parallel axis theorem for rod about new CM

163

$$I_{cm} = \left\{ \frac{m M^2}{(m+M)^2} + \frac{1}{3} M + \frac{M m^2}{(m+M)^2} \right\} \frac{l^2}{4}$$

$$= \frac{3 M m + M (m+M)}{m+M} \frac{l^2}{12} = \frac{(4m+M)M}{m+M} \frac{l^2}{12}$$

$$\omega = \frac{L_{int}}{I_{cm}} = \frac{m+M}{(4m+M)M} \frac{12}{l^2} \times \frac{m M v}{m+M} \frac{l}{2}$$

$$= \frac{6m}{4m+M} \frac{v}{l}$$



A bowling ball of mass M and radius

R is released moving at velocity v_0 and begins to slide with coefficient of friction μ . What is its final velocity

$$M \frac{dv}{dt} = -\mu M g \quad v(t) = v_0 - \mu g t$$

$$I \dot{\omega} = \tau = \mu M g R$$

$$I_{cm} = \left\{ \frac{mM^2}{(m+M)^2} + \frac{1}{3}M + \frac{Mm^2}{(m+M)^2} \right\} \frac{l^2}{4}$$

$$= \frac{3Mm + M(m+M)}{m+M} \frac{l^2}{12} = \frac{(4m+M)M}{m+M} \frac{l^2}{12}$$

$$\omega = \frac{L_{int}}{I_{cm}} = \frac{m+M}{(4m+M)M} \frac{12}{l^2} \times \frac{mMv}{m+M} \frac{l}{2}$$

$$= \frac{6m}{4m+M} \frac{v}{l}$$



A bowling ball of mass M and radius

R is released moving at velocity v_0 and begins to slide with coefficient of friction μ . What is its final velocity

$$M \frac{dv}{dt} = -\mu Mg \quad v(t) = v_0 - \mu g t$$

$$I \dot{\omega} = \tau = \mu Mg R$$

$$\dot{\omega} = \frac{\mu Mg R}{I} = \frac{\mu Mg R}{\frac{2}{5}MR^2} = \frac{5\mu g}{2R}$$

$$\omega(t) = \frac{5\mu g}{2R} t$$

stops sliding when

$$R\omega(t) = v(t)$$

$$\frac{5}{2} \mu g t = v_0 - \mu g t$$

$$\frac{7}{2} \mu g t = v_0, \quad t = \frac{2}{7} \frac{v_0}{\mu g}$$

$$v = \omega R = \frac{5\mu g}{2} \frac{2}{7} \frac{v_0}{\mu g} = \frac{5}{7} v_0$$